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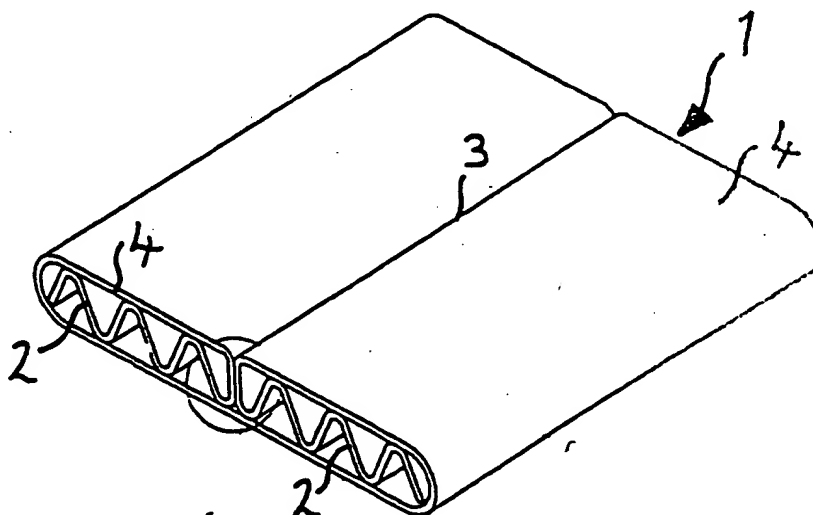
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INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

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<p>(21) International Application Number: PCT/GB93/01332 (22) International Filing Date: 24 June 1993 (24.06.93) (30) Priority data: 9213358.6 24 June 1992 (24.06.92) GB (71) Applicant (for all designated States except US): LLANELLI RADIATORS LIMITED [GB/GB]; Llanelli, Dyfed SA14 8HU (GB). (72) Inventor; and (75) Inventor/Applicant (for US only): YUKITAKE, Taizo [JP/GB]; 12 Dan-y-Lan, Felinfoel, Llanelli, Dyfed SA14 8BW (GB). (74) Agent: AUSTIN, Hedley, William; Urquhart-Dykes & Lord, Alexandra House, Alexandra Road, Swansea, West Glamorgan SA1 5ED (GB).</p>		<p>(81) Designated States: JP, US, European patent (AT, BE, C, DE, DK, ES, FR, GB, GR, IE, IT, LU, MC, NL, SE). Published With international search report.</p>

(54) Title: HEAT EXCHANGE TUBES



(57) Abstract

A heat exchange tube (1) comprises, in its formed state, an outer wall (4) surrounding a plurality of internal fins (2) which extend along the length of the tube. The fins (2) and outer wall (4) are formed from a unitary portion of sheet material, the portion comprising respective groups of fins extending from a common longitudinal seam line (3) in mutually opposed directions transverse to the longitudinal direction of the tube and seam. The tube (1) is typically formed by means of a roll forming process in which the groups of fins (2) are initially formed in the sheet or strip and portions of the strip subsequently plastically deform longitudinally about a longitudinal axis of the sheet or strip to provide the outer wall. Typically the tube is formed from sheet

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According to a first aspect of the invention, there is provided a heat exchange tube comprising an outer wall surrounding a plurality of internal fins extending longitudinally of the tube, the fins and outer wall being formed from a unitary portion of sheet or strip material, the fins comprising respective groups of fins each group comprising a respective shaped portion of the sheet or strip material, the groups of fins extending from a common longitudinal seam line in mutually opposed directions which opposed directions are transverse to the longitudinal direction of the tube.

The common longitudinal seam line comprises a line of abutment of respective portions of the wall of the tube which are inverted during forming to position the groups of fins internally of the tube.

Typically the common longitudinal seam line comprises a bonded join, typically a brazed join.

It is preferred that a pair of groups of fins are provided, advantageously extending transversely from the seam line to substantially the same extent such that in transverse cross-section the tube is preferably substantially symmetrical about the seam line.

Desirably, the shaped portions of the strip or sheet material defining each group of fins are preferably separated from one another by interconnecting portions, which interconnecting portions are not provided with fins.

Advantageously, the groups of fins are provided each adjacent a respective longitudinally running peripheral edge of the sheet or strip material.

The tube is required to be heat conductive, and therefore the strip or sheet material from which the tube is formed is typically of metal or alloy. It is preferred that the strip or sheet material comprises clad aluminium to aid in the brazing of the tube and also the brazing of the final heat exchanger assembly. Portions of the fins are typically brazed to respective portions of the outer wall to improve the thermal conductive connection therebetween.

According to a second aspect, the invention therefore comprises a heat exchanger having one or more heat exchange tubes as defined herein.

In use, the heat exchange tubes are arranged for flow of heat transfer fluid therethrough from an inlet to an outlet spaced therefrom along a fluid flow path between the inlet and outlet defined by the tube.

Advantageously, the outer surface profile of the tube is arranged such that effectively two substantially parallel external heat exchange surfaces are provided. It is preferred that the width of the heat exchanger tube is substantially greater than its thickness.

Typically, each group of fins in the interior of the tube are corrugated, comprising alternating troughs and crests in thermally conductive contact with respective opposed portions of the outer wall. In a similar manner, the corrugated fins may comprise castellations or any other suitable configuration having fin surfaces extending between opposed portions of the outer wall of the tube. In a preferred embodiment the corrugated fins are provided with louvres or slits such that fluid may pass through the surfaces of the corrugated fins. Typically, the corrugated fins define a plurality of longitudinally extending fluid flow pathways along the interior of the tube.

Typically, the heat exchange tube is formed by a roll forming process, and therefore, according to a third aspect, the invention comprises a method of forming a heat exchange tube comprising forming respective groups of fins in respective deformable portions of strip or sheet material, and subsequently deforming further portions of the strip or sheet material to provide an outer wall surrounding the groups of fins, whereby the groups of fins extend from a common longitudinal seam line in mutually opposed directions which directions are transverse to the longitudinal direction of the seam line.

Desirably, two respective groups of fins are provided, each in the region of a respective longitudinally running edge of the strip or sheet material.

Advantageously, subsequently to formation of the groups of fins, the sheet material is deformed symmetrically about a longitudinal axis to form the heat exchange tube.

It is preferred that the portions of the sheet material provided with respective groups of fins are folded (typically by roll forming) toward one another causing intermediate portions of the sheet or strip material to wrap around the groups of fins thereby providing the outer wall.

Typically, the tube is then brazed along the seam line to form a joining interface between the respective groups of fins.

The invention will now be further described in a specific embodiment by way of example only and with reference to the accompanying drawings, in which:

Figures 1 to 3 show known heat exchange tubes of various constructions;

Figure 4 shows an initial stage in the formation of a heat exchanger tube according to the invention;

Figures 5 and 6 show successive intermediate stages in the formation of a heat exchanger tube according to the invention;

Figure 7 shows a section of finished heat exchanger tube according to the invention;

Figure 8 shows a preferred embodiment of a part of the heat exchanger tube shown in Figure 7; and

Figure 9 is a schematic representation of apparatus arranged to form the finished heat exchanger tube shown in Figure 7.

Referring initially to Figures 1 to 3, various types of known (prior art) heat exchanger tubes are shown. Figure 1 shows a tube 13 which comprises an outer wall 14 roll formed from clad aluminium strip which is then brazed along a longitudinal edge. A fin corrugated insert 15 is subsequently inserted into the tube and brazed to give a good thermal connection to the outer wall 14.

Referring to Figure 2, there is shown an extruded heat exchange tube 16 which is extruded integrally from aluminium billet stock. Fins 17 are formed integrally with the outer wall 18 during extrusion. Referring to Figure 3, there is shown a typical oil cooler heat exchange tube 19 extruded from billet stock.

Referring now to Figures 4 to 9 which relate to the present invention, there is shown a section of elongate heat exchanger tube generally designated 1. The tube shown is suitable for use in heat exchangers such as vehicle radiators, condensers, oil coolers, inter-coolers, heaters etc. where heat is to be transferred between a first fluid medium carried in the interior of tube 1 (usually at a relatively high temperature for radiators and oil coolers) and a second fluid medium which passes over the exterior surfaces of the tube (usually at a relatively lower temperature for radiators and oil coolers).

The tube 1 is formed integrally from a single initially flat strip of clad aluminium by a roll forming process (described below) such that integral corrugated fins 2 are formed in the interior of the tube 1. The tube is then brazed (typically in unison with the remainder of the assembled heat exchanger) using a known brazing process to give a single longitudinal brazed tube join along longitudinal seam 3 and give good brazed thermally conductive connection between the crests and troughs of the corrugated fins 2 and the interior of the outer surrounding tube wall 4.

Referring to Figure 9, a continuous clad aluminium strip 11 is fed from a reel 5 into the first station of multistation roll forming apparatus 6. Typically, the roll forming apparatus 6 has between 10 and 40 stations, each station typically comprising pairs of rolls arranged to symmetrically plastically deform respective portions of the aluminium strip to a predetermined pattern or configuration. For example, an initial series of roll stations will be arranged to successively deform the longitudinal peripheral portions of the strip to provide respective series of corrugated fins 2 shown in Figure 4 (only one peripheral portion is shown in Figures 4 and 5). Intermediate stations in the roll forming apparatus 6 successively deform the strip to the configurations shown in Figures 5 and 6 until, on leaving the roll forming apparatus 6, the configuration of the strip has been deformed to that shown in Figure 7 which is the finished configuration of the tube. Because the aluminium strip is arranged to be deformed to the required configuration symmetrically about its longitudinal axis 20, the manufacturing process using the series of "in-line" roll forming stations 6 is particularly convenient. It is therefore possible to conveniently form an effectively continuous heat exchange tube from unitary sheet with integrally formed internal fins. Because the tube 1 is symmetrical about the brazed seam 3, the integrity and rigidity of the tube is also maximised.

On leaving the roll forming apparatus 6 the continuous tube is cut to the required length at a cutting station 7 before being carried on conveyor 8 to a heat exchanger jig 9 in which the cut to length tubes 1 are placed alternately with layers of concenterad fins 10 (which define the second fluid flow matrix) before the assembled heat exchanger is brazed in a single brazing operation.

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Referring to Figure 8, certain stations in the roll forming apparatus may be provided with perforating means arranged to produce perforated louvres or slits 12 in the corrugated fins 2. The louvres 12 increase the turbulence of the fluid medium carried in the tube 1, and hence increases the heat transfer efficiency between the two fluid media.

CLAIMS:

1. A heat exchange tube comprising an outer wall surrounding a plurality of internal fins extending longitudinally of the tube, the fins and outer wall being formed from a unitary portion of sheet or strip material, the fins comprising respective groups of fins each group comprising a respective shaped portion of the sheet or strip material, the groups of fins extending from a common longitudinal seam line in mutually opposed directions which opposed directions are transverse to the longitudinal direction of the tube.
2. A heat exchange tube according to claim 1, wherein the common longitudinal seam line comprises a bonded join.
3. A heat exchange tube according to claim 1 or claim 2, wherein in transverse cross-section the tube is substantially symmetrical about the seam line.
4. A heat exchange tube according to any preceding claim, wherein the shaped portions of the strip or sheet material defining each group of fins are separated from one another by interconnecting portions, which interconnecting portions are not provided with fins.
5. A heat exchange tube according to any preceding claim, wherein the groups of fins are formed each adjacent a respective longitudinally running peripheral edge of the sheet or strip material.
6. A heat exchange tube according to any preceding claim, wherein each group of fins in the interior of the tube are corrugated, comprising alternating troughs and crests in thermally conductive contact with respective opposed portions of the outer wall.
7. A heat exchange tube according to any preceding claim, wherein the fins are provided with louvres or slits such that fluid may pass through the surfaces of the fins.

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8. A heat exchange tube according to any preceding claim wherein the strip or sheet material comprises clad aluminium or clad aluminium alloy.
9. A heat exchanger having one or more heat exchange tubes according to any preceding claim.
10. A method of forming a heat exchange tube comprising forming respective groups of fins in respective deformable portions of strip or sheet material, and subsequently deforming further portions of the strip or sheet material to provide an outer wall surrounding the groups of fins, whereby the groups of fins extend from a common longitudinal seam line in mutually opposed directions which directions are transverse to the longitudinal direction of the seam line.
11. A method according to claim 10, wherein the sheet material is deformed symmetrically about its longitudinal axis to form the heat exchange tube.
12. A method according to claim 10 or claim 11, wherein two respective groups of fins are formed, each in the region of a respective longitudinally running peripheral edge of the strip or sheet material.
13. A method according to any of claims 10 to 12, wherein the portions of the sheet material are provided with respective groups of fins are folded toward one another causing intermediate portions of the sheet or strip material to wrap around the groups of fins thereby providing the outer wall.
14. A method according to any of claims 10 to 13, wherein the tube is brazed along the seam line to form a joining interface between the respective groups of fins.

Figure 1

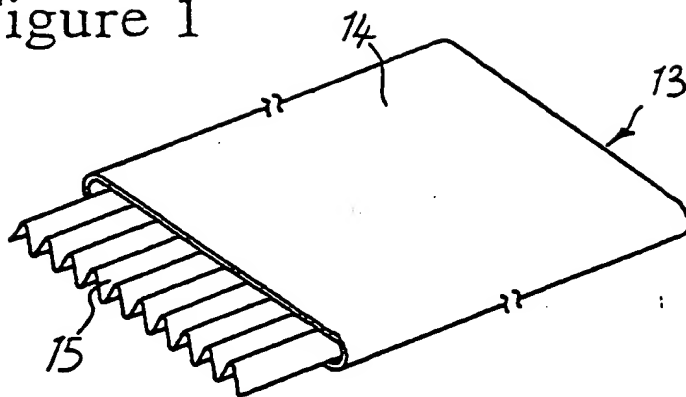


Figure 2

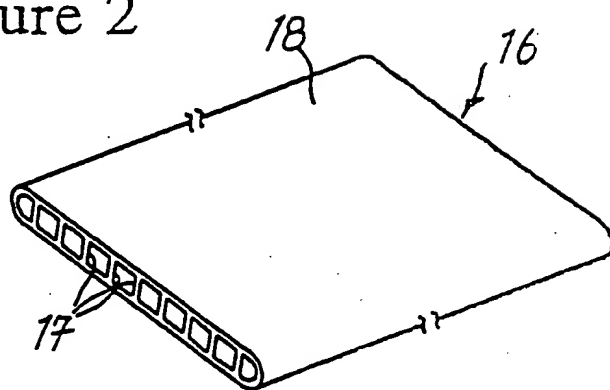


Figure 3

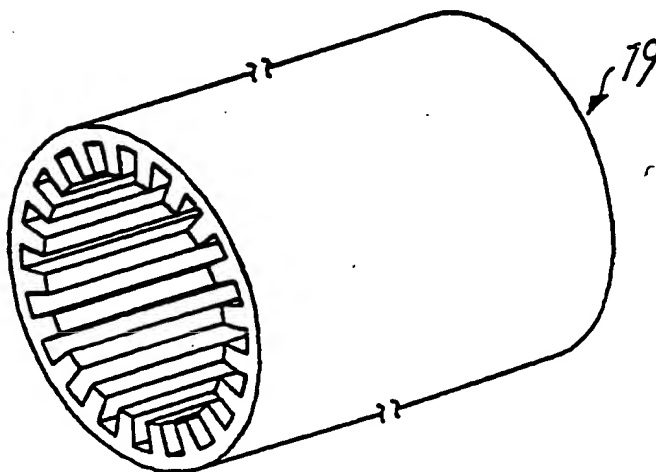


Figure 4

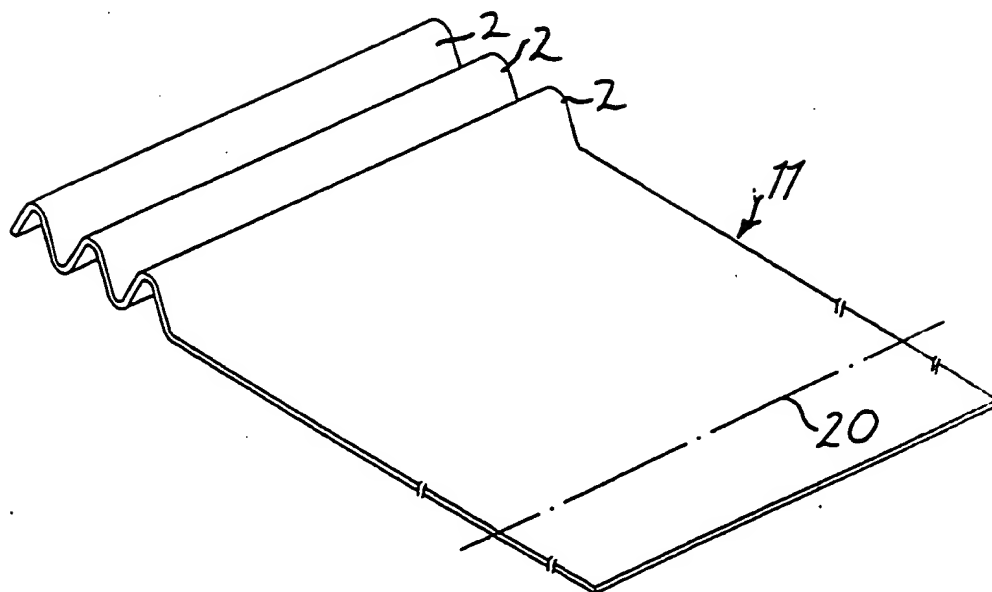


Figure 5

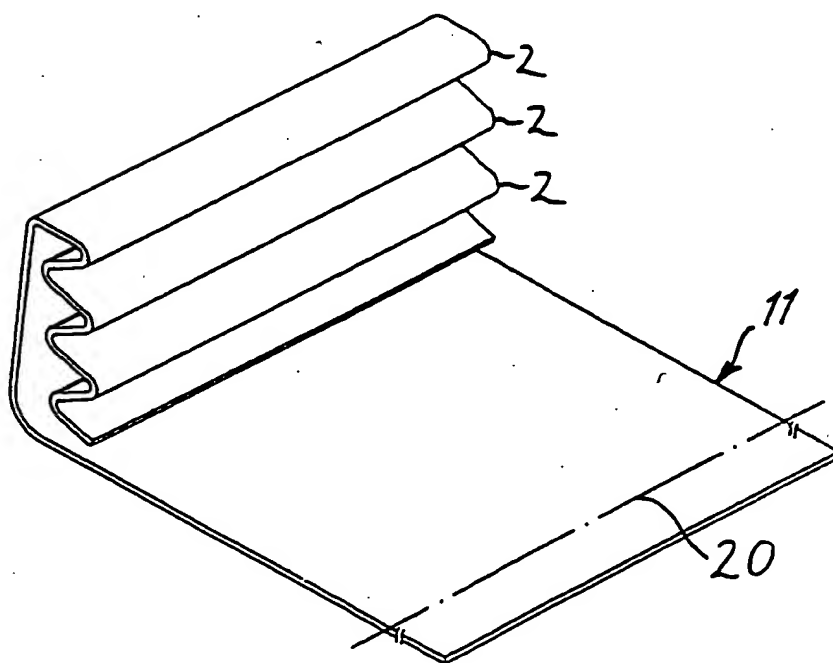


Figure 6

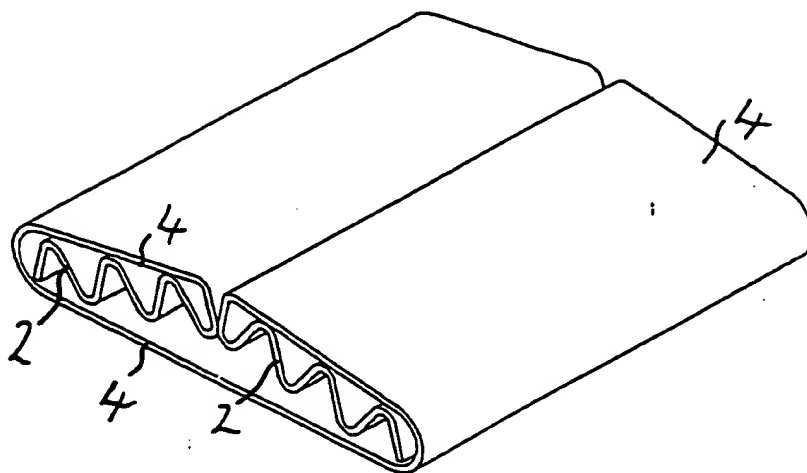


Figure 7

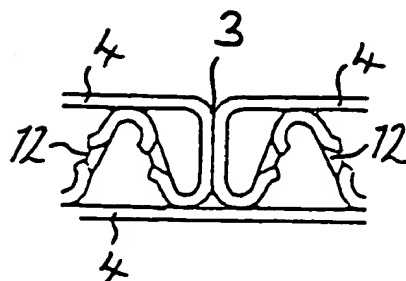
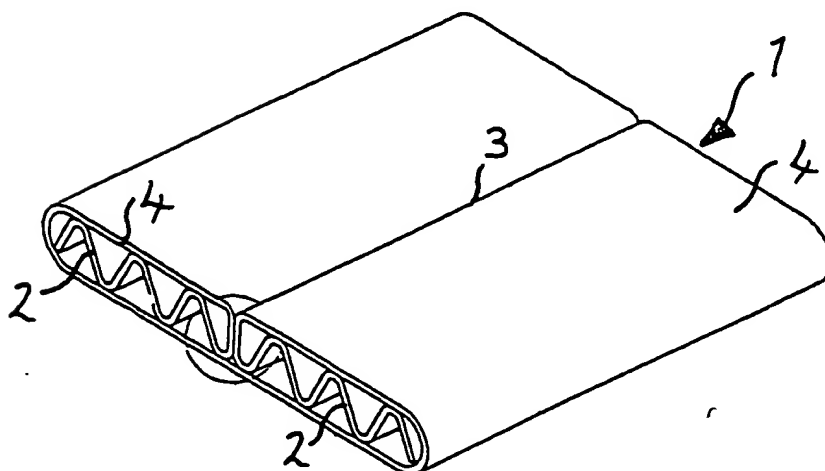
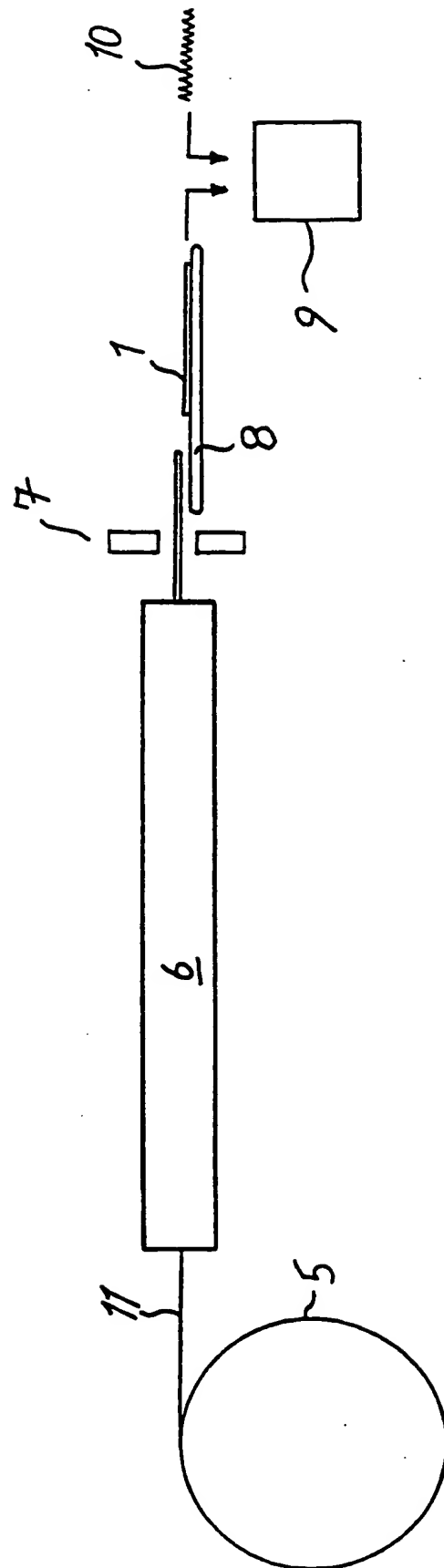


Figure 8

Figure 9



INTERNATIONAL SEARCH REPORT

International Application No.

PCT/GB 93/01332

I. CLASSIFICATION OF SUBJECT MATTER (If several classification symbols apply, indicate all) ⁶ According to International Patent Classification (IPC) or to both National Classification and IPC IPC ⁵ : F 28 F 1/40, F 28 D 1/00		
II. FIELDS SEARCHED <div style="text-align: center; border-top: 1px solid black; border-bottom: 1px solid black; margin: 5px 0;">Minimum Documentation Searched ⁷</div> <div style="display: flex; justify-content: space-between;"> <div style="width: 30%; border-bottom: 1px solid black; padding-bottom: 5px;">Classification System </div> <div style="width: 70%; border-bottom: 1px solid black; padding-bottom: 5px;">Classification Symbols</div> </div> <div style="padding: 5px 0;"> IPC⁵ : F 28 F 1/00, F 28 D 1/00, B 21 C 37/00 </div> <div style="border-top: 1px solid black; padding-top: 5px; margin-top: 10px;"> Documentation Searched other than Minimum Documentation to the extent that such Documents are included in the Fields Searched ⁸ </div>		
III. DOCUMENTS CONSIDERED TO BE RELEVANT ⁹		
Category ¹⁰	Citation of Document, ¹¹ with indication, where appropriate, of the relevant passages ¹²	Relevant to Claim No. ¹³
A	EP, A1, 0 429 166 (WALLIS) 29 May 1991 (29.05.91), fig. 2,7. <div style="text-align: center;">--</div>	1,9,10
A	GB, A, 1 601 954 (COVRAD) 04 November 1981 (04.11.81), fig. 1; claims 1-3. <div style="text-align: center;">--</div>	1,2,3, 6,9,10
A	US, A, 4 351 392 (STOCKMAN) 28 September 1982 (28.09.82), fig. 3. <div style="text-align: center;">--</div>	1,9,10
A	US, A, 3 902 552 (MCLAIN) 02 September 1975 (02.09.75), fig. 6,8. <div style="text-align: center;">----</div>	1,9,10
<div style="display: flex; justify-content: space-between;"> <div style="width: 45%;"> <p>¹⁴ Special categories of cited documents: ¹⁵</p> <p>¹⁶ "A" document defining the general state of the art which is not considered to be of particular relevance</p> <p>¹⁷ "E" earlier document but published on or after the international filing date</p> <p>¹⁸ "L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)</p> <p>¹⁹ "O" document referring to an oral disclosure, use, exhibition or other means</p> <p>²⁰ "P" document published prior to the international filing date but later than the priority date claimed</p> </div> <div style="width: 45%;"> <p>²¹ "T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention</p> <p>²² "X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step</p> <p>²³ "Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art</p> <p>²⁴ "Z" document member of the same patent family</p> </div> </div>		
IV. CERTIFICATION		
Date of the Actual Completion of the International Search <div style="text-align: center; font-size: 1.2em;">22 September 1993</div>		Date of Mailing of this International Search Report <div style="text-align: center; font-size: 1.2em;">18. 10. 93</div>
International Searching Authority <div style="text-align: center; font-weight: bold;">EUROPEAN PATENT OFFICE</div>		Signature of Authorized Officer <div style="text-align: center; font-weight: bold;">HUBER_e.h.</div>

ANHANG

zum internationalen Recherchen-
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ANNEX

to the International Search
Report to the International Patent
Application No.

ANNEXE

au rapport de recherche inter-
national relatif à la demande de brevet
international n°

PCT/GB 93/01332 SAE 76803

In diesem Anhang sind die Mitglieder
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This Annex lists the patent family
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cited in the above-mentioned inter-
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In Recherchenbericht angeführtes Patentdokument Patent document cited in search report Document de brevet cité dans le rapport de recherche	Datum der Veröffentlichung Publication date Date de publication	Mitglied(er) der Patentfamilie Patent family member(s) Membre(s) de la famille de brevets	Datum der Veröffentlichung Publication date Date de publication
EP A1 429166	29-05-91	JP A2 3169427 US A 4971240	23-07-91 20-11-90
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US A 4351392	28-09-82	CA A1 1158637 JP A2 57129397	13-12-83 11-08-82
US A 3902552	02-09-75	US A 3831675 US A 3901430 US A 3918626 US A 3921883 US A 3885622 US A 3858785 US A 3861462 US A 3906605 AU A1 68613/74 CA A1 1005365 DE A1 2422340 FR A1 2228550 FR B1 2228550 GB A 1448901 GB A 1448902 IT A 1021543 JP A2 50027754 JP A2 56037495 SE A 7406020	27-08-74 26-08-75 11-11-75 25-11-75 27-05-75 07-01-75 21-01-75 23-09-75 06-11-75 15-02-77 21-11-74 06-12-74 21-11-80 08-09-76 08-09-76 20-02-78 22-03-75 11-04-81 31-01-75